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(54) **CHAIN CAPTURE ASSEMBLY FOR CHAIN TENSIONERS**

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(\*) Notice: Subject to any disclaimer, the term of this  
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(65) **Prior Publication Data**

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**F16G 11/12** (2006.01)

**B60P 7/06** (2006.01)

(57) **ABSTRACT**

Aspects of the invention are directed to an apparatus that may be used as a chain tensioner. The apparatus comprises a release handle, which is limited in motion by an elongate member that rides in a shaped guide slot. As a result of this configuration, two distinct forces must be applied to the release handle before a chain is released from the apparatus. Such a configuration reduces the chances of inadvertent releases of a chain.

(52) **U.S. Cl.**

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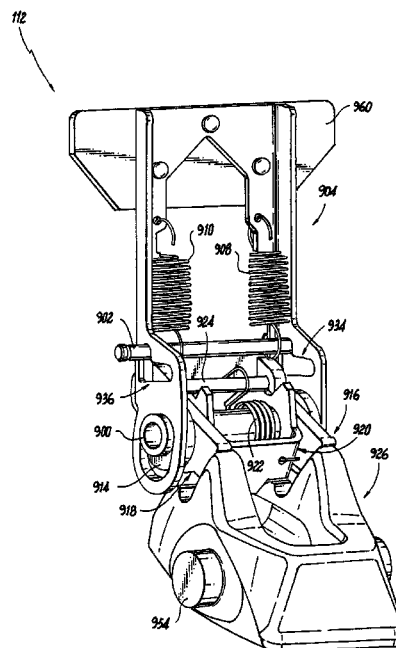
(58) **Field of Classification Search**

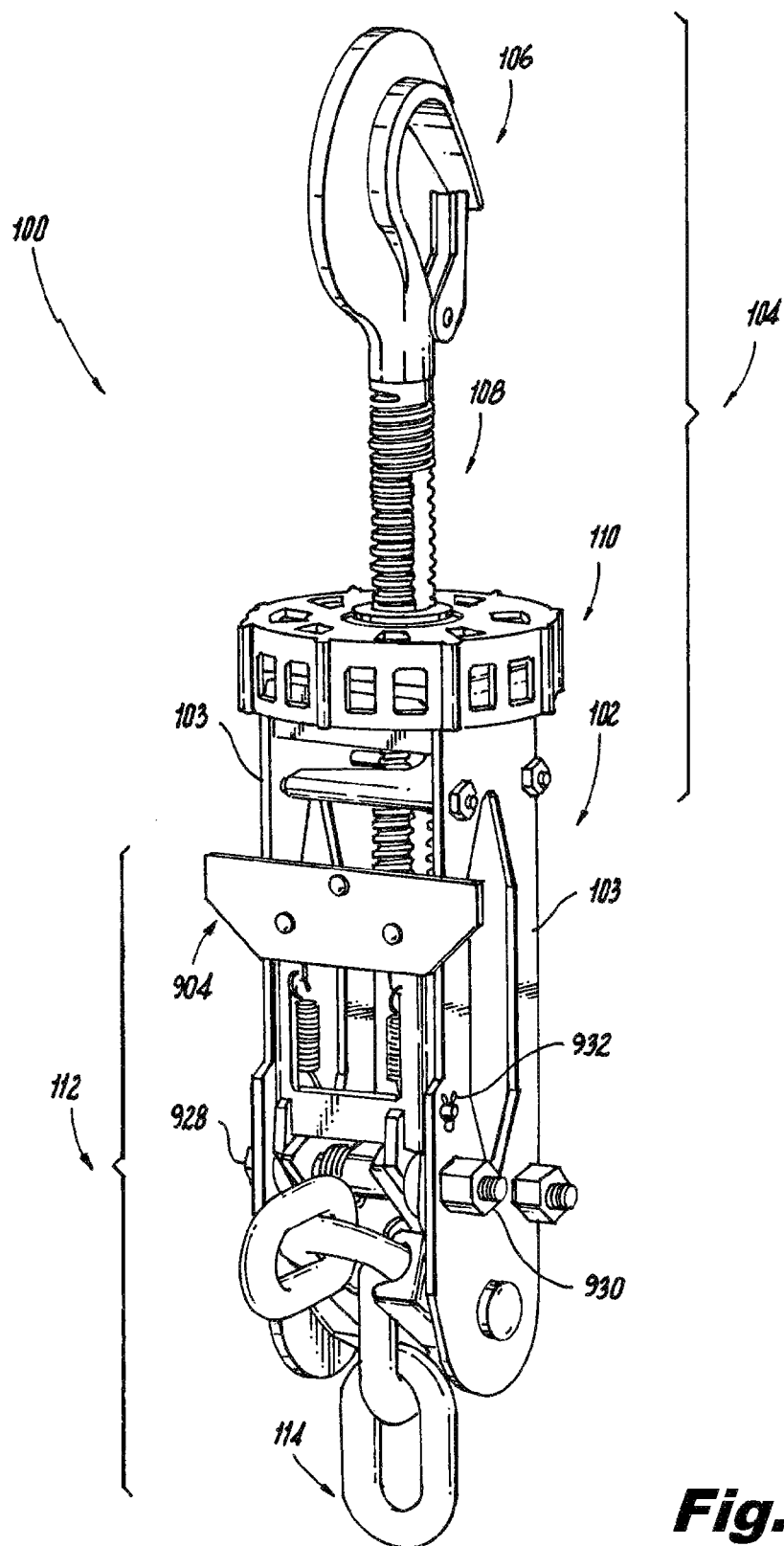
CPC ..... B60P 7/0823; B60P 7/083

USPC ..... 410/96, 97, 100

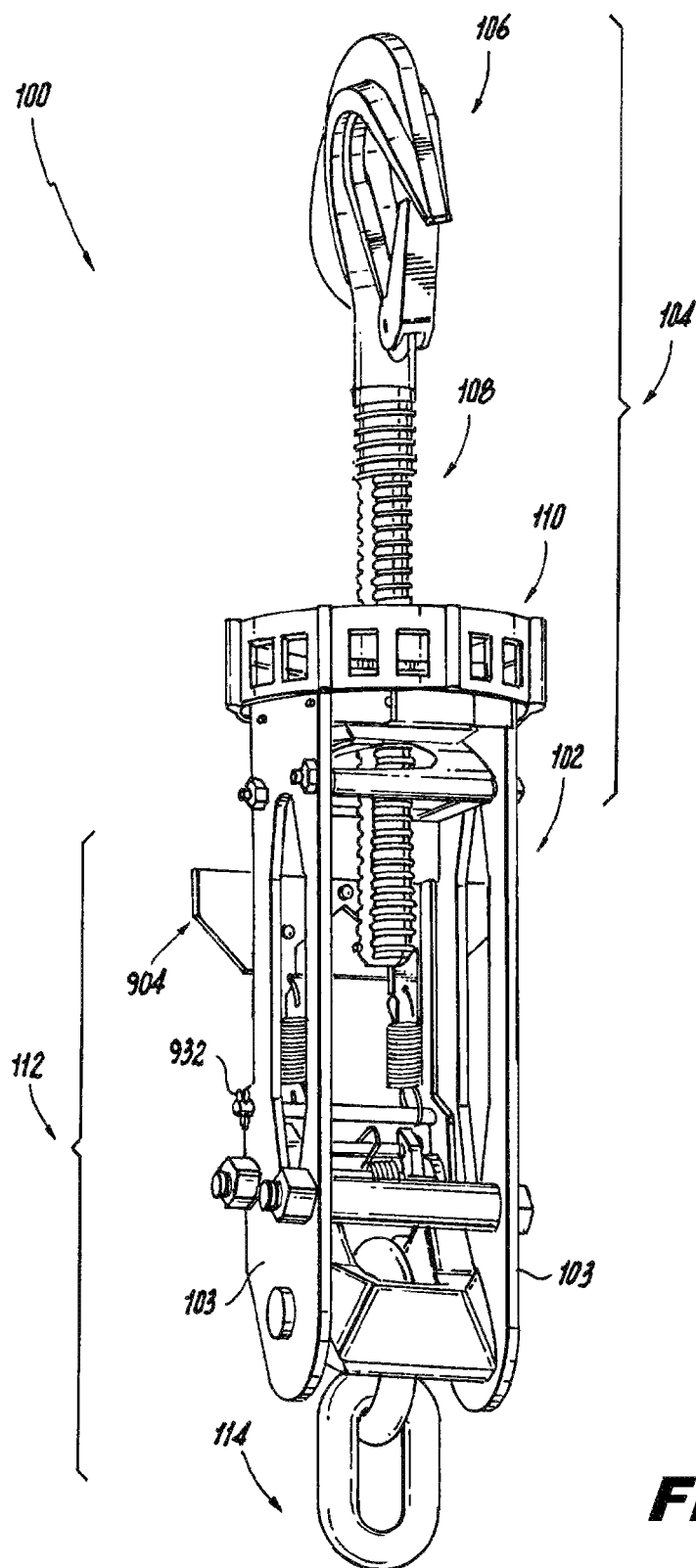
See application file for complete search history.

**20 Claims, 14 Drawing Sheets**

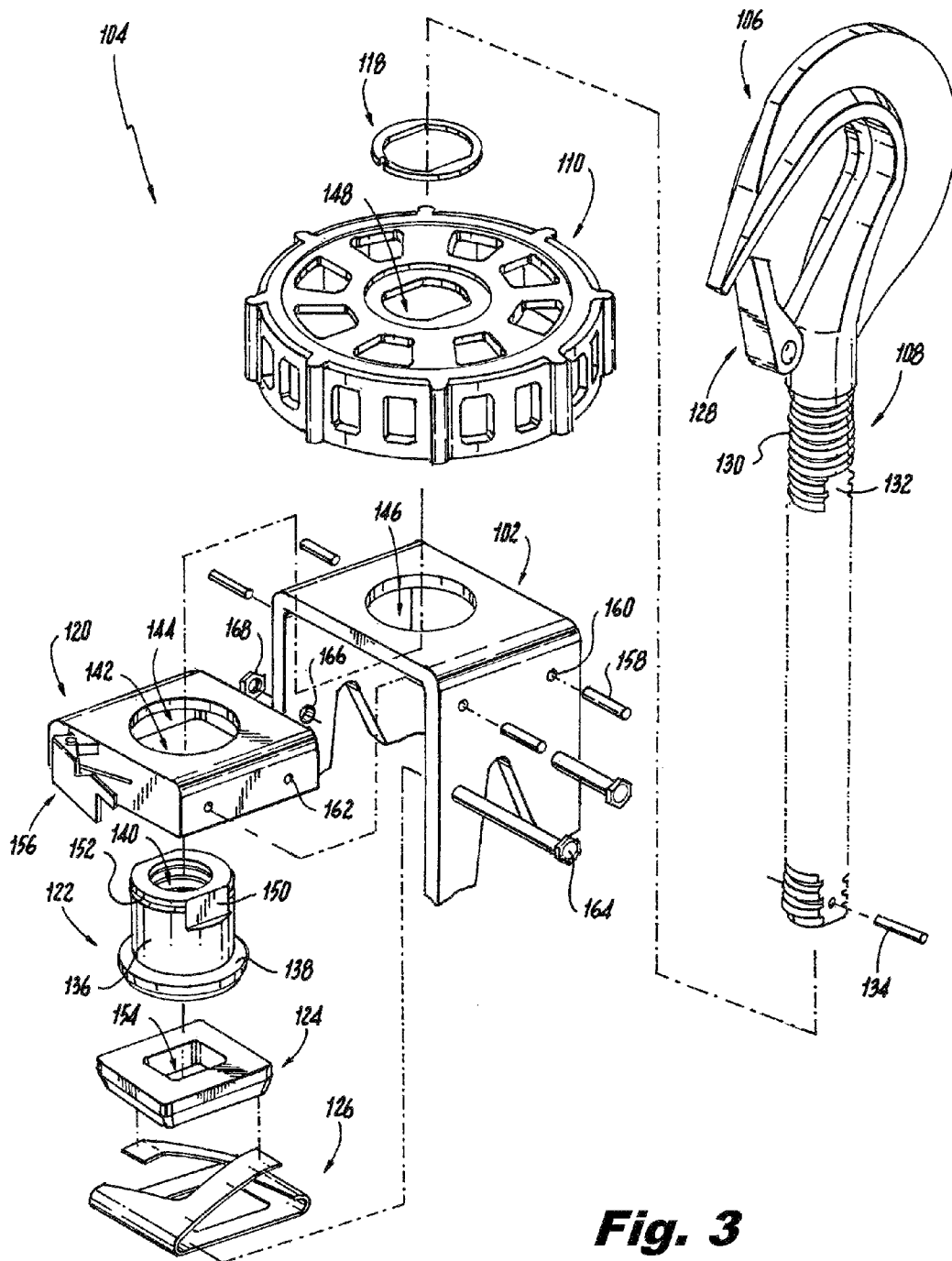




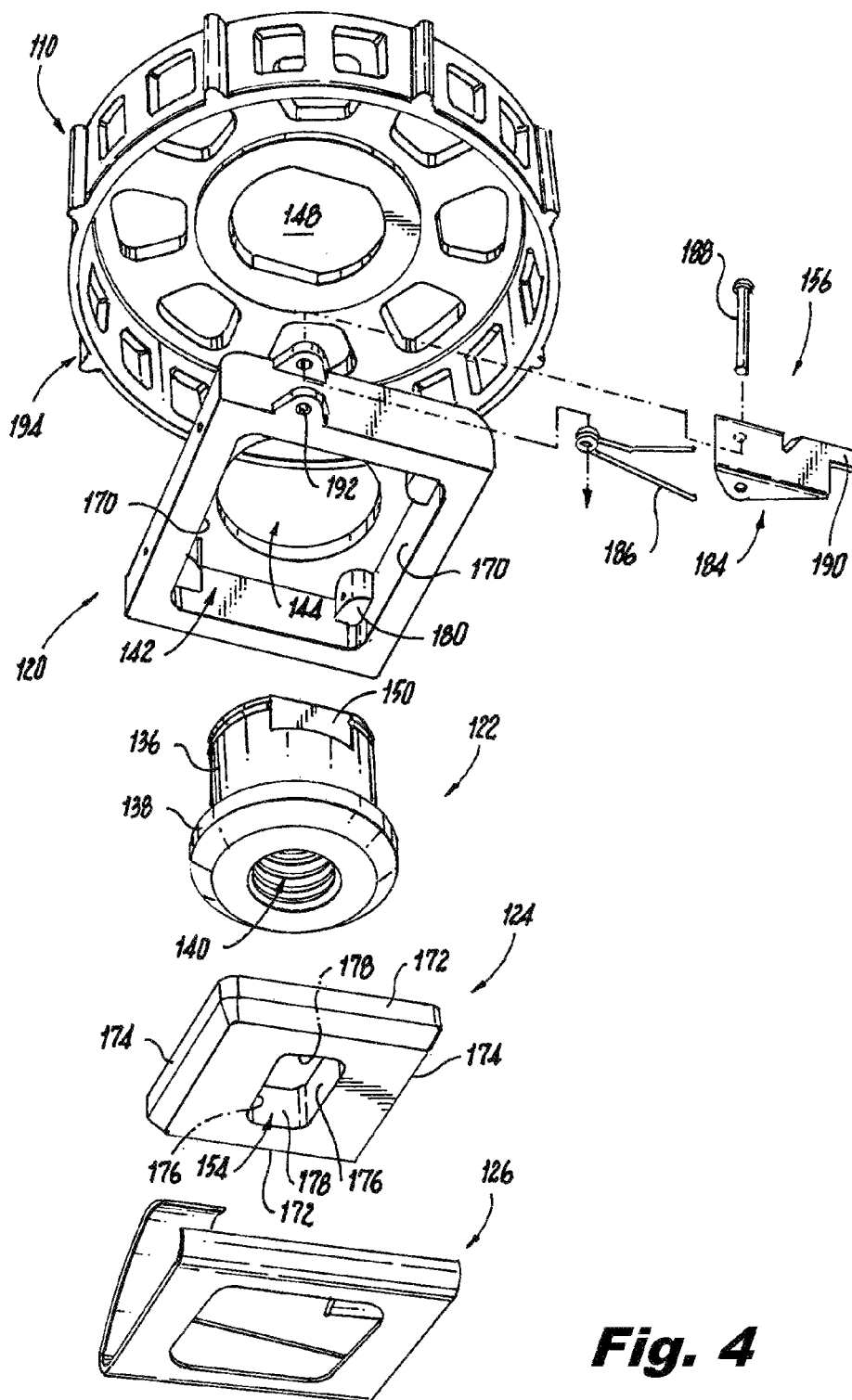
**Fig. 1**



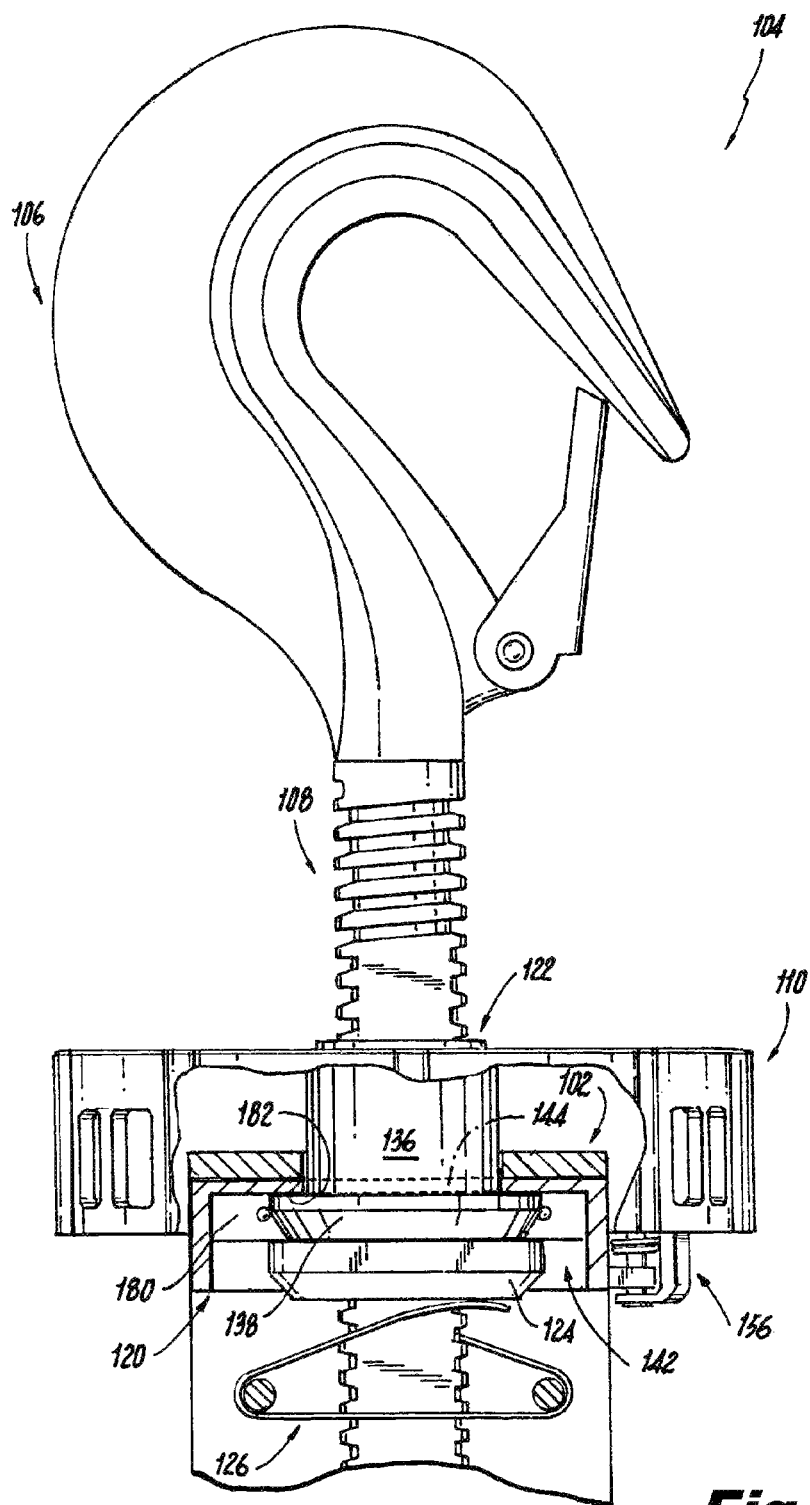
**Fig. 2**



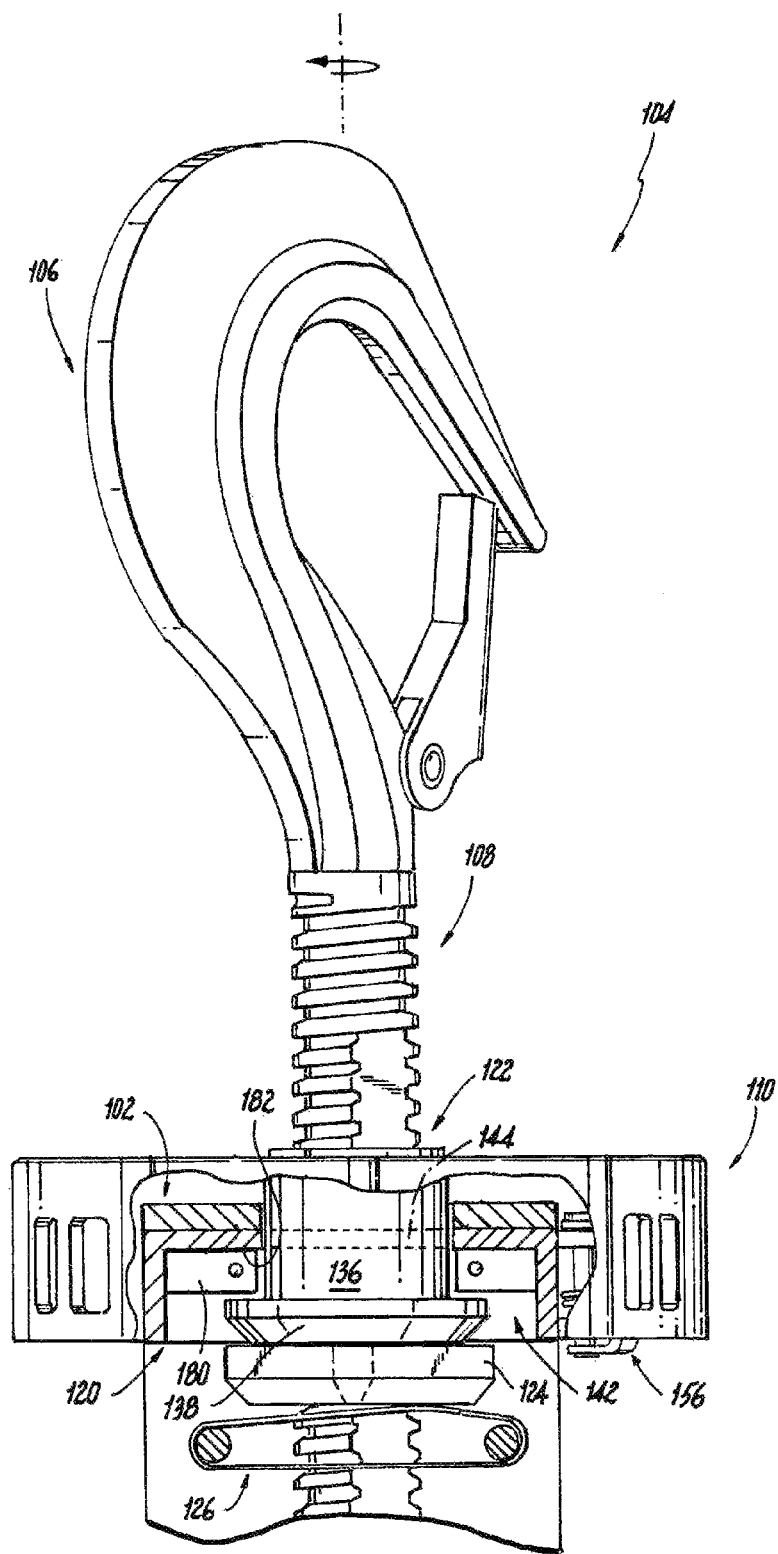
**Fig. 3**



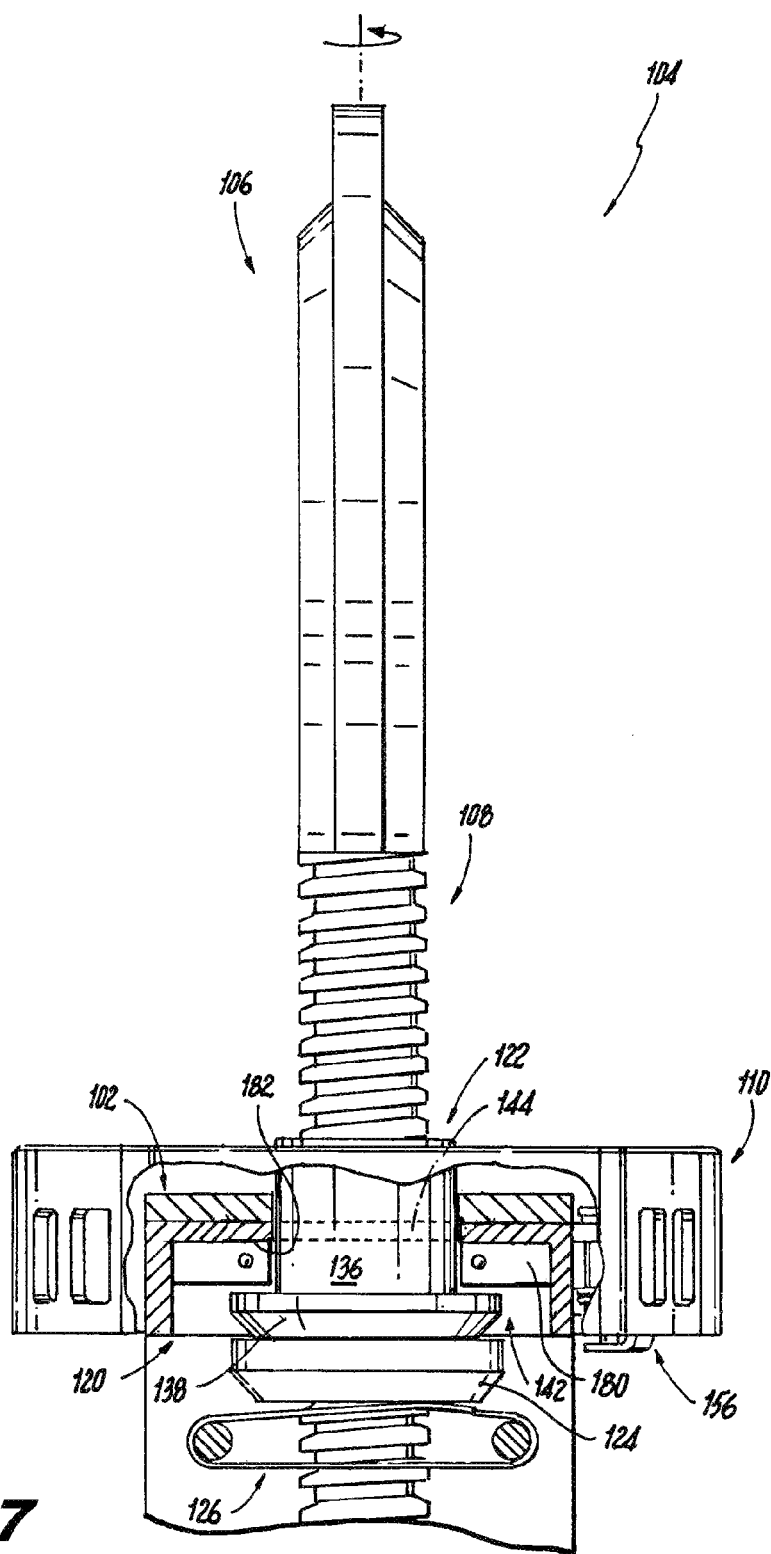
**Fig. 4**



**Fig. 5**



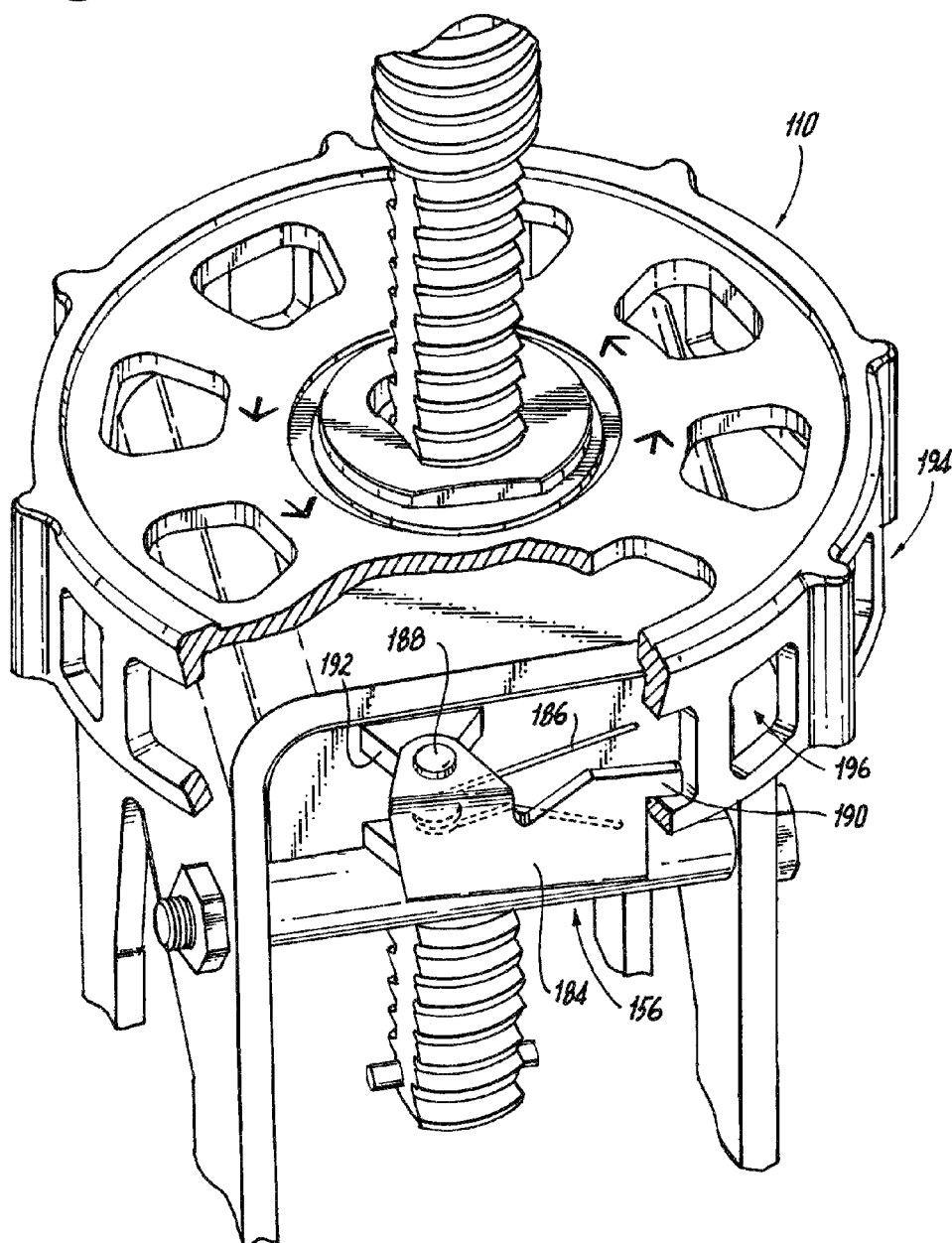
**Fig. 6**

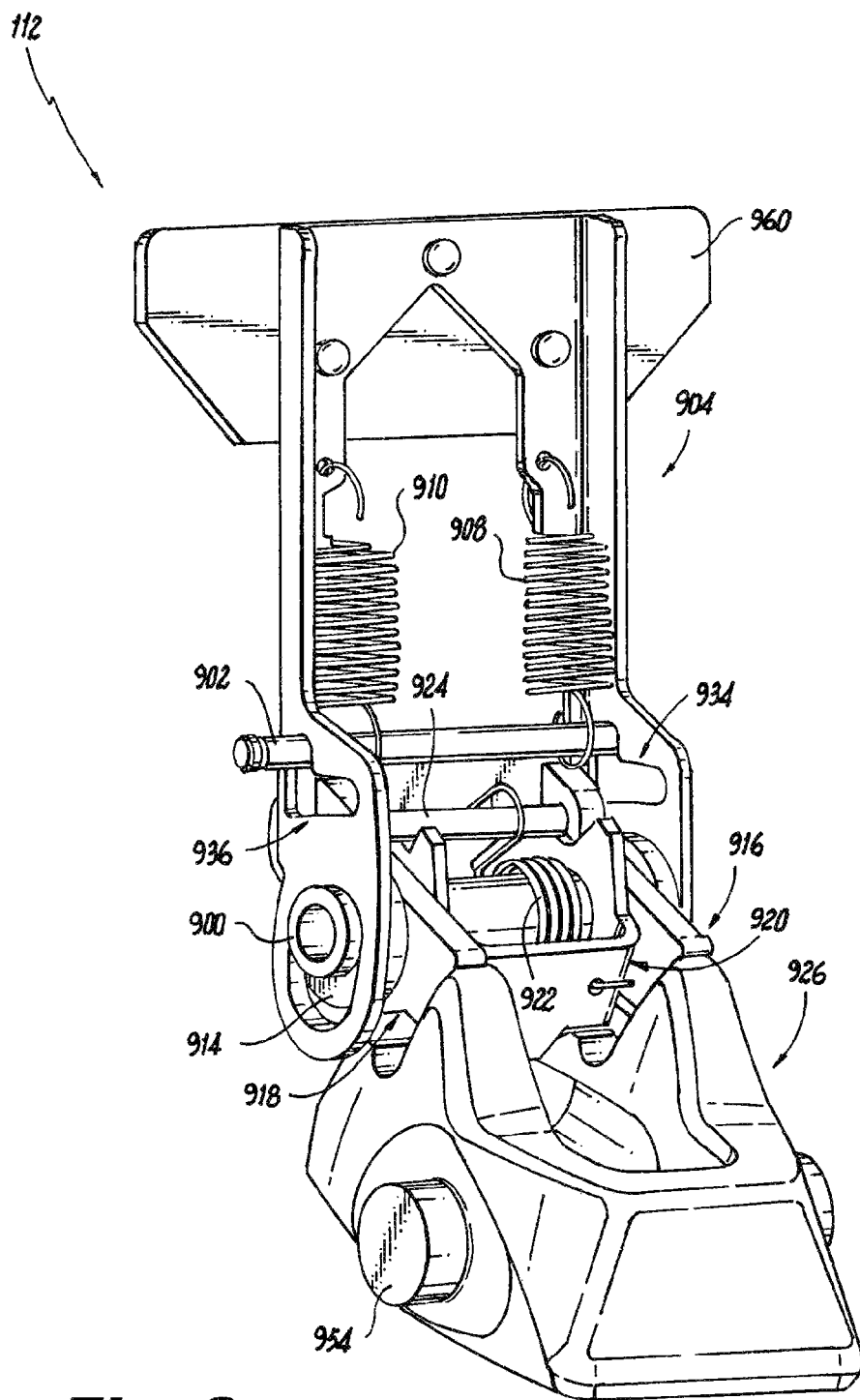


**Fig. 7**

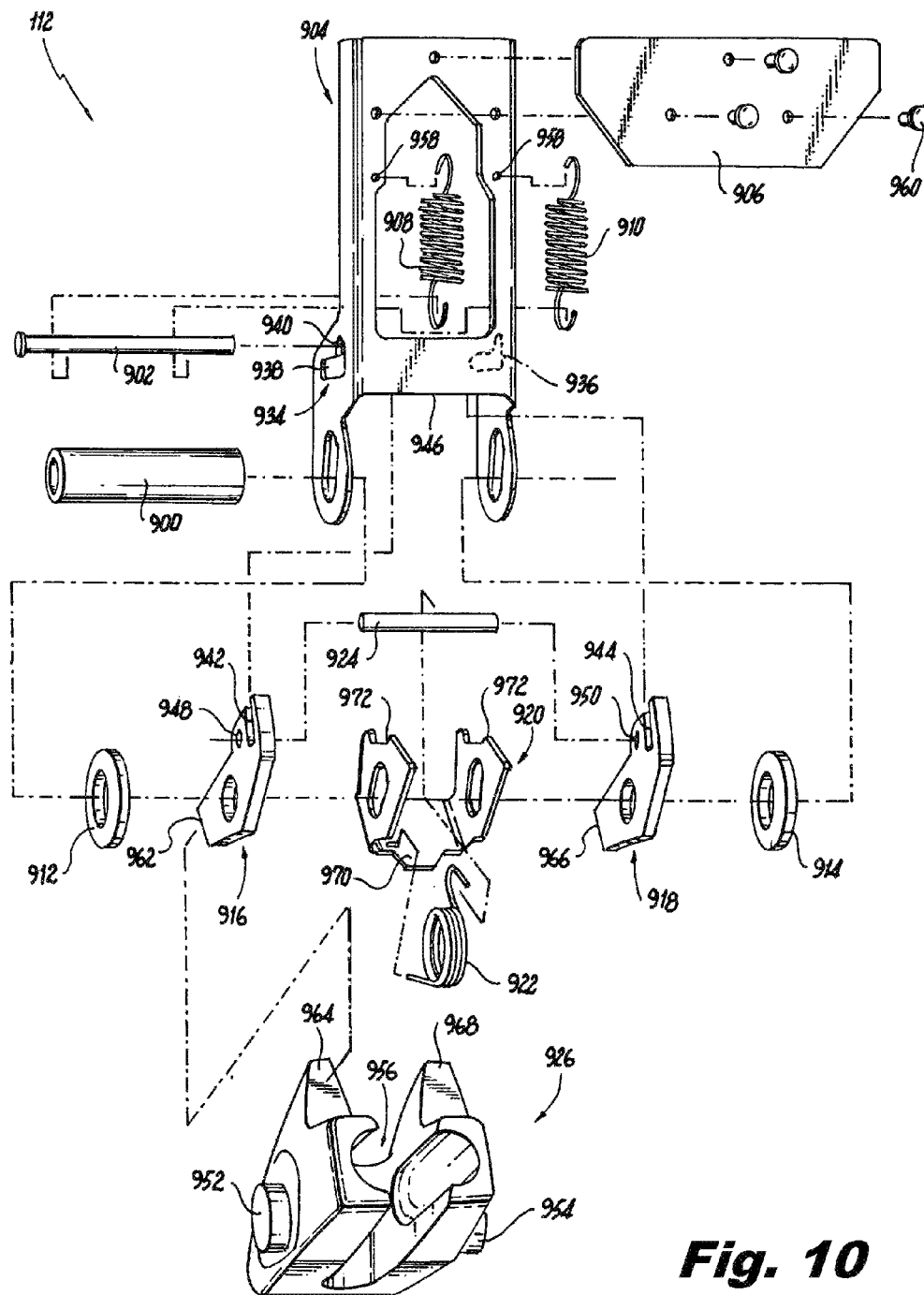


**Fig. 8**

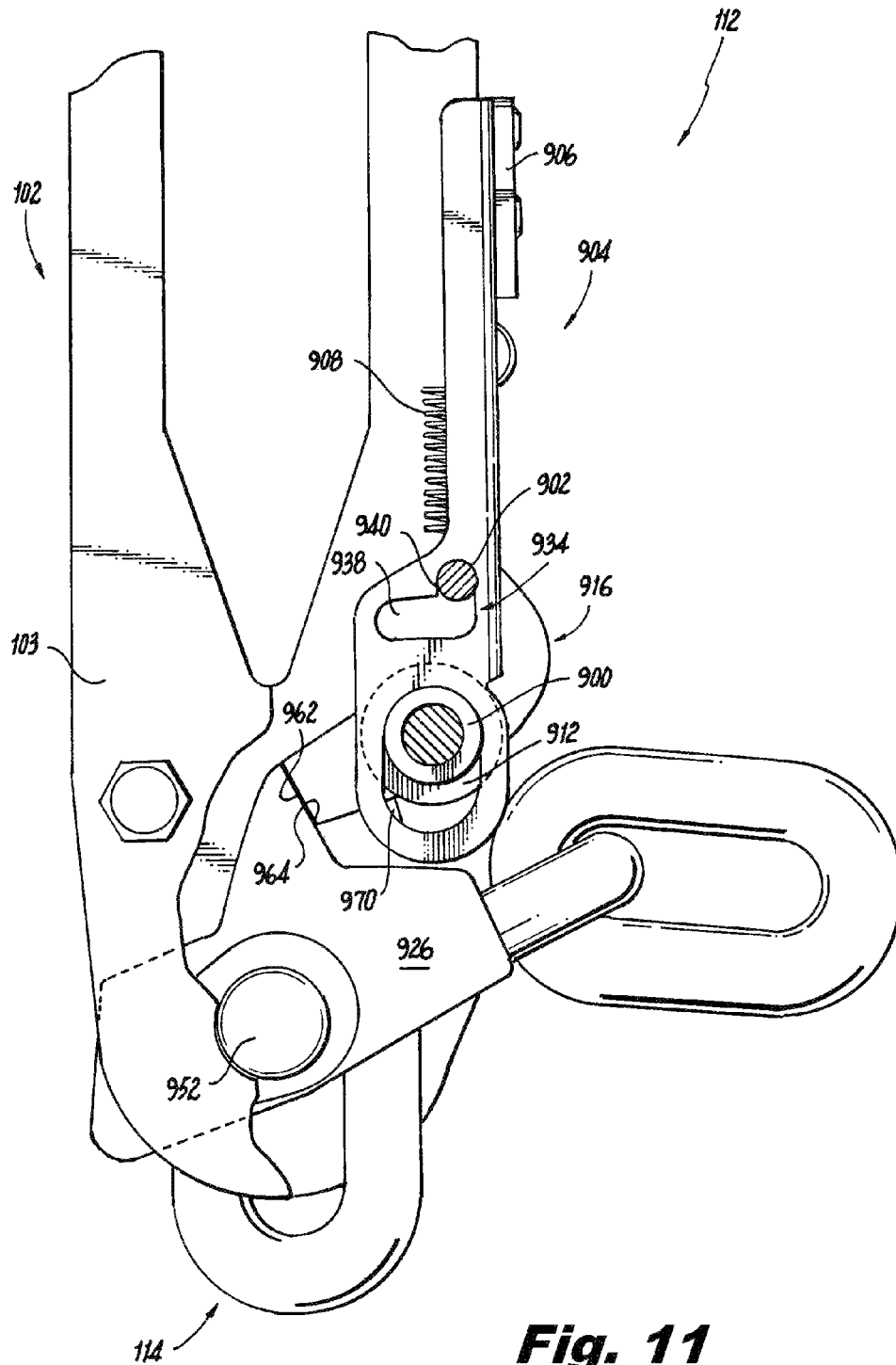




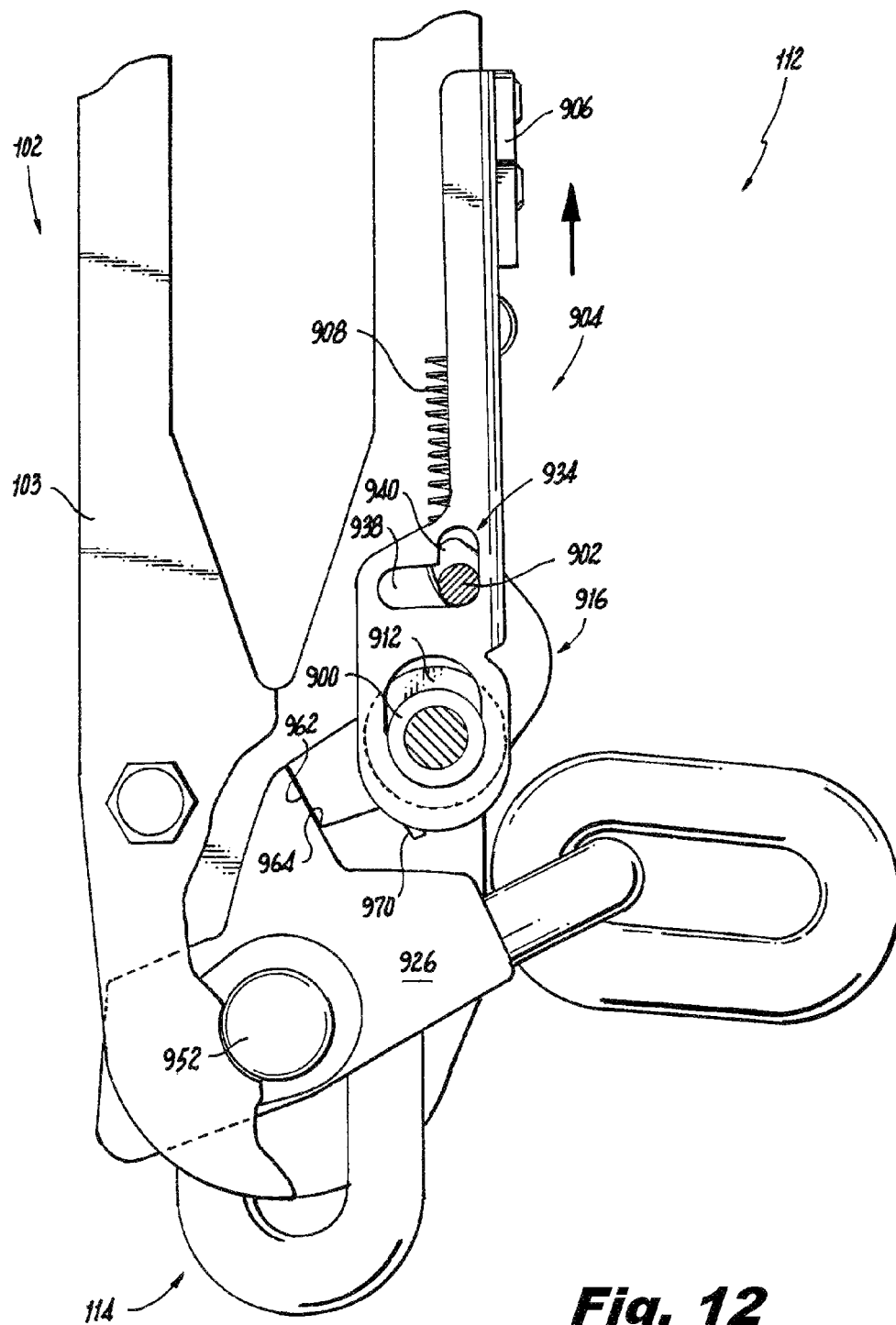
**Fig. 9**



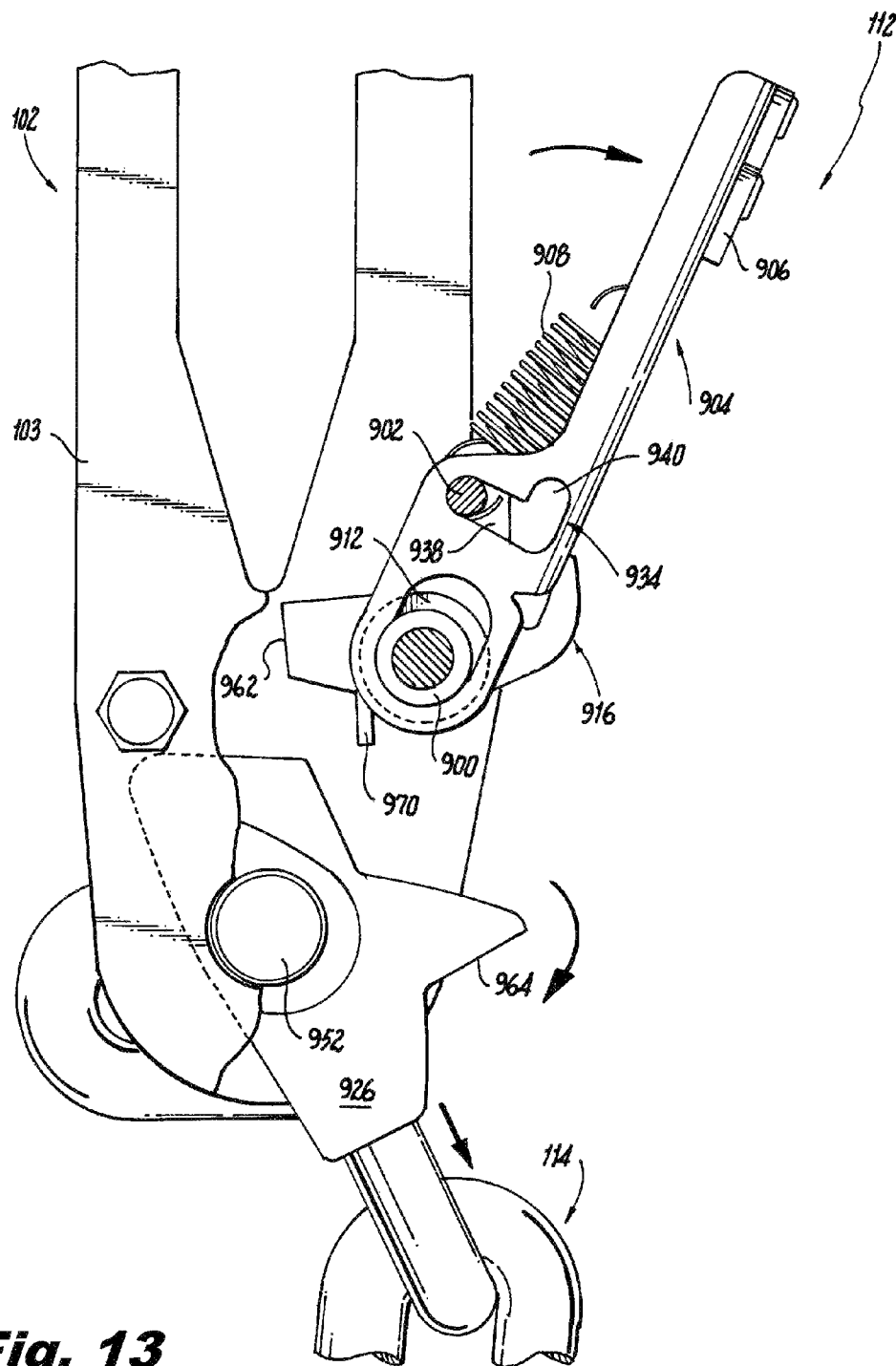
**Fig. 10**



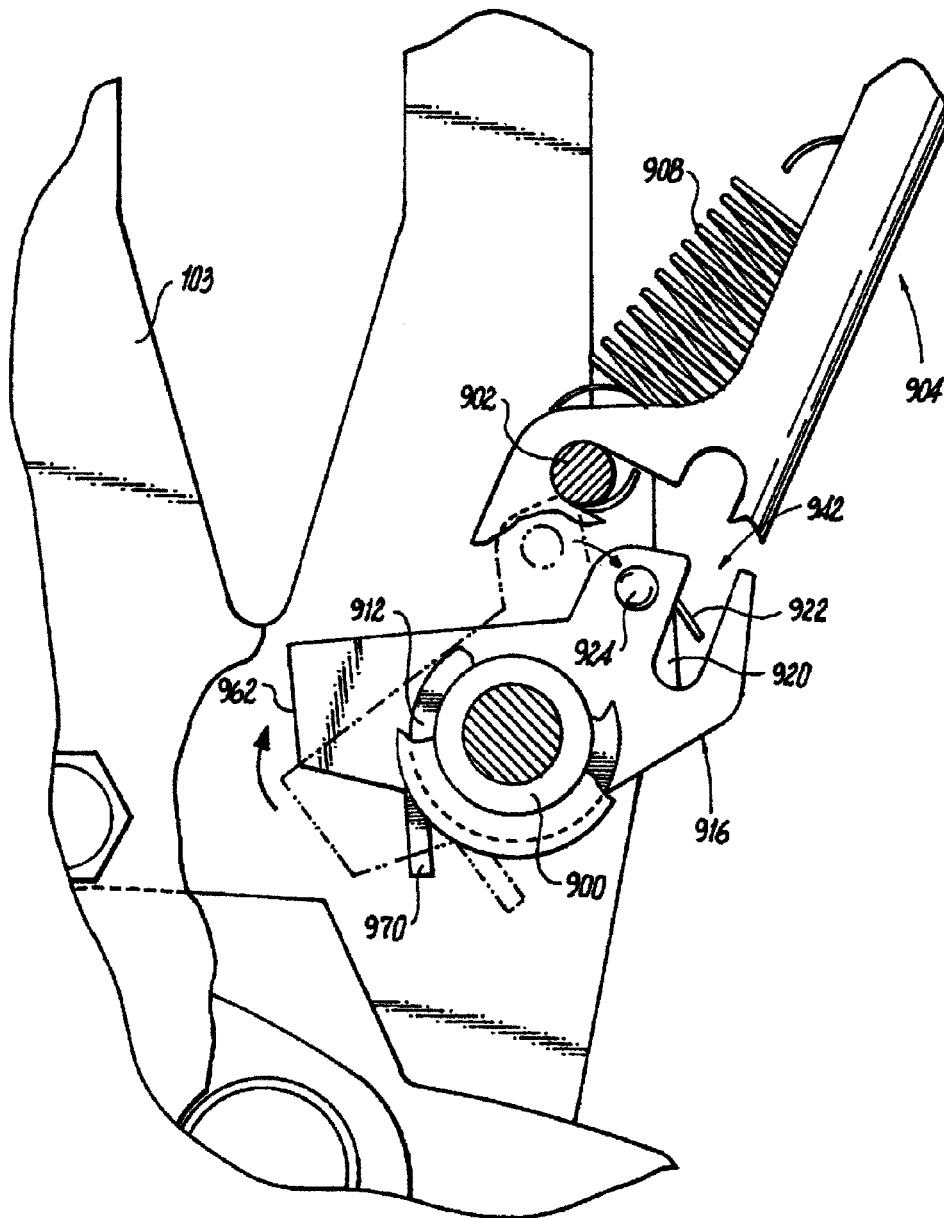
**Fig. 11**



**Fig. 12**



**Fig. 13**



**Fig. 14**

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## CHAIN CAPTURE ASSEMBLY FOR CHAIN TENSIONERS

### FIELD OF THE INVENTION

The present invention relates generally to load-engaging elements, and, more particularly, to chain tensioners for use in securing cargo for transport.

### BACKGROUND OF THE INVENTION

Chain tensioners (sometimes called “tie down tensioners”) are commonly used to secure cargo during transport. A chain tensioner will typically comprise an apparatus with a hook at one end and a means for attaching a chain at the other. The chain may be attached to the cargo, while the hook may be attached to a tie-down ring on an aircraft or other form of transport. After the chain is initially attached between the chain tensioner and the cargo, a tensioning wheel on the chain tensioner may be rotated to effectively reduce the length of the chain tensioner, and thereby increase the tension on the chain. Later, the chain may be quickly released by manipulating a release handle on one side of the chain tensioner. Such chain tensioners are available from, for example, Davis Aircraft Products Co. (Bohemia, NY, USA), as well as others.

An unintended release of a chain by a chain tensioner is a very real hazard when utilizing these devices. Personnel moving about an aircraft may, for example, accidentally kick the release handle of a chain tensioner while that chain tensioner is actively securing cargo. If such an unintended release were to occur in an aircraft, a shift in cargo could suddenly place the center-of-gravity in a position where the aircraft is no longer controllable. In November 2011, the U.S. Department of Defense responded to this danger by publishing a specification for military chain tensioners that explicitly specifies: “The mechanism [for chain release] shall be designed in such a manner that inadvertent release of the mechanism by personnel moving about the aircraft does not occur” (U.S. Detail Specification MIL-DTL-26959H, 13 Nov. 2012).

For the foregoing reasons, there is a need for new chain tensioner designs that incorporate chain connection assemblies that minimize the chances of inadvertent releases.

### SUMMARY OF THE INVENTION

Embodiments of the present invention address the above-identified needs by providing chain tensioner designs that incorporate a release handle that must be actuated by two distinct forces before allowing a chain to be released. Such a configuration reduces the chances the chain will be released inadvertently.

Aspects of the invention are directed to an apparatus comprising: a frame, a primary elongate member, a release handle, a secondary elongate member, a rotatable member, and a chain capture block. The primary elongate member is fixated to the frame, while the release handle is pivotally supported by the primary elongate member and defines a shaped guide slot. The secondary elongate member is fixated to the frame, and rides in the shaped guide slot. The rotatable member is pivotally supported by the primary elongate member, and is pivotally coupled to the release handle such that the rotatable member pivots in unison therewith. Lastly, the chain capture block defines a cavity therein shaped to accommodate one or more links of a chain, and is pivotally mounted to the frame such that the chain capture block can achieve a chain capture orientation and a chain release orientation relative to the frame. In use, the shaped guide slot and the secondary

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elongate member cooperate to allow the release handle to be translated relative to the frame between a locked configuration and an unlocked configuration. Moreover, the shaped guide slot and the secondary elongate member cooperate to allow the release handle to be pivoted relative to the frame between the unlocked configuration and an open configuration. Finally, the rotatable member interferes with the pivoting of the chain capture block from the chain capture orientation to the chain release orientation when the release handle is in the locked or unlocked configuration, and the chain capture block is free to pivot between the chain capture orientation and the chain release orientation when the release handle is in the open configuration.

### BRIEF DESCRIPTION OF THE DRAWINGS

These and other features, aspects, and advantages of the present invention will become better understood with regard to the following description, appended claims, and accompanying drawings where:

FIGS. 1 and 2 show front and rear perspective views, respectively, of an apparatus 100 in accordance with an illustrative embodiment of the invention;

FIG. 3 shows an exploded side perspective view of the hook assembly in the apparatus 100, in accordance with an illustrative embodiment of the invention;

FIG. 4 shows a bottom perspective view of several elements within the FIG. 3 hook assembly;

FIGS. 5-7 show partially cutaway elevational views of a portion of the FIG. 3 hook assembly in locked and unlocked states;

FIG. 8 shows a partially cutaway perspective view of a portion of the FIG. 3 assembly relating to the tension wheel locking mechanism;

FIG. 9 shows a rear perspective view of the chain capture assembly in the apparatus 100, in accordance with an illustrative embodiment of the invention;

FIG. 10 shows a front exploded perspective view of elements in the FIG. 9 chain capture assembly;

FIGS. 11-13 show partially broken side elevational views of the FIG. 9 chain capture assembly and chain with the chain capture assembly in the locked, unlocked, and open configurations, respectively; and

FIG. 14 shows another partially broken side elevational view of the FIG. 9 chain capture assembly and chain with the chain capture assembly in the unlocked configuration.

### DETAILED DESCRIPTION OF THE INVENTION

The present invention will be described with reference to illustrative embodiments. For this reason, numerous modifications can be made to these embodiments and the results will still come within the scope of the invention. No limitations with respect to the specific embodiments described herein are intended or should be inferred.

As used herein, the term “about,” when used to modify a length, shall mean within 15 percent of the recited length. In addition, as used herein, pairs of sidewalls, surfaces, or edges that are “opposed,” “parallel,” or “oppositely facing” run parallel to each other within ten degrees.

FIGS. 1 and 2 show front and rear perspective views, respectively, of an apparatus 100 in accordance with an illustrative embodiment of the invention. The apparatus may be conceptually separated into two portions, which are mated together by a u-shaped frame 102 defining a pair of parallel sidewalls 103. At the top of the apparatus 100 (as oriented in FIGS. 1 and 2), a hook assembly 104 provides a hook 106 and



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a threaded shaft **108**, which are longitudinally translatable relative to the frame **102** in response to the rotation of a tensioning wheel **110**. At the bottom of the apparatus, a chain capture assembly **112** provides a means for releasably attaching a chain **114** to the apparatus **100**.

So configured, the apparatus **100** may be utilized as a chain tensioner device to aid in securing cargo for transport in an aircraft or other mode of transport. Cargo that is attached to the chain **114**, for example, can be secured for transport by attaching the hook **106** at one end of the apparatus **100** to a tie-down ring, and then attaching the chain **114** to the apparatus **100** at the other end. The apparatus thereby becomes a bridge between the tie-down ring and the chain **114**. The tensioning wheel **110** may subsequently be rotated to obtain the desired tension across the chain **106** and the apparatus **100**. It is contemplated that embodiments in accordance with aspects of the invention would be capable of applying tensions in excess of 300 pounds. Once the cargo is successfully transported, the chain can be quickly disconnected via a release handle **904**.

Each of the hook assembly **104** and the chain capture assembly **112** comprises novel and non-obvious aspects, and each portion is now addressed in detail. Embodiments in accordance with aspects of the invention may comprise elements of both portions, or may only comprise aspects of one of the two portions.

#### I. Hook Assembly

FIG. **3** shows an exploded side perspective view of the hook assembly **104** in the apparatus **100**, in accordance with an illustrative embodiment of the invention. In addition to the frame **102**, the hook **106**, the threaded shaft **108**, and the tensioning wheel **110**, the hook assembly **104** further comprises: a retaining ring **118**, a locking body **120**, a threaded receiver **122**, a locking plate **124**, and a biasing element **126**. The hook **106** comprises a spring-loaded keeper **128**. The threaded shaft **108**, in turn, attaches to the base of the hook **106** and extends in the longitudinal direction therefrom. The threaded shaft **108** defines two different profiles along its length. An upper portion of the threaded shaft proximate to the hook **106** is defined by an externally threaded cylinder **130**. Further down the threaded shaft **108**, the threaded shaft **108** is defined by the externally threaded cylinder **130** with a pair of oppositely facing flat surfaces **132**. Near the bottom of the threaded shaft, a pin **134** passes through the threaded shaft **108** to stop the threaded shaft **108** from being overly extended.

The threaded receiver **122** defines an upper cylindrical portion **136** and a lower protruding lip **138** that surround a threaded passageway **140**. At the same time, the locking body **120** defines a cavity **142** and an aperture **144** that is in communication with the cavity **142**. When assembled, the lower protruding lip **138** of the threaded receiver **122** occupies the cavity **142**, while the upper cylindrical portion **136** passes through the aperture **144**. After so doing, the upper cylindrical portion **136** further passes through a lower opening **146** in the frame **102** before ultimately engaging a shaped opening **148** in the tensioning wheel **110**. Where the upper cylindrical portion **136** engages the shaped opening **148**, the upper cylindrical portion **136** defines a pair of outside flat surfaces **150** that match the shape of the shaped opening **148**. Above the tensioning wheel **110**, the retaining ring **118** clips onto the upper cylindrical portion at a groove **152**. The tensioning wheel **110** and the threaded receiver **122** are thereby both rotationally and translatably coupled to one another. That is, rotating the tensioning wheel **110** causes the threaded receiver **122** to also rotate. Moving the tensioning wheel **110**

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up and down along the longitudinal axis causes the threaded receiver **122** to also move up and down along the same axis.

The threaded passageway **140** in the threaded receiver **122** threadably engages the threaded shaft **108**. After passing through the threaded receiver **122**, the threaded shaft **108** further extends through a mating slot **154** in the locking plate **124** and through the biasing element **126**. The biasing element **126** presses the locking plate **124** against the threaded receiver **122**. In the present illustrative embodiment, the biasing element **126** defines a leaf spring in contact with the locking plate **124**, but this particular arrangement is not intended as being limiting. A tensioning wheel locking mechanism **156** is also visible on an outside face of the locking body **120**, and is further detailed below.

In the present illustrative embodiment, the locking body **120** is fixated in the frame **102** by four roll pins **158** that pass through frame apertures **160** in the frame **102** and engage complementary holes **162** in the locking body **120**. The biasing element **126** is fixated to the frame **102** by two bolts **164** that pass through bolt openings **166** in the frame **102** while simultaneously passing through the biasing element **126**. The bolts are secured by nuts. **168**

The tensioning wheel **110**, the locking body **120**, the threaded receiver **122**, the locking plate **124**, and the biasing element **126** cooperate to allow the hook **106** to be both translated inward and outward along the longitudinal axis of the apparatus **100**, as well as swiveled between, and locked into, multiple orientations relative to the frame **102**. FIG. **4** shows an exploded bottom perspective view of these elements so that additional aspects that lend to this functionality may be more easily appreciated.

It will be noted, for example, that the locking body **120** and the locking plate **124** are dimensionally related to one another. In the present illustrative embodiment, the cavity **142** in the locking body **120** is rectangular and defines a pair of parallel internal sidewalls **170** along its narrower, width axis. At the same time, the locking plate **124**, which, in this example is square, defines two pairs of parallel outside edges **172**, **174**. In accordance with aspects of the invention, the pair of parallel internal sidewalls **170** are spaced apart by a distance about equal to the spacing between the two pairs of parallel outside edges **172**, **174**. It is therefore possible to insert the locking plate **124** into the cavity **142** of the locking body **120** in four different orientations. Once so inserted, the pair of parallel internal sidewalls **170** of the locking body **120** interact with one of the pairs of parallel outside edges **172**, **174** of the locking plate **124** to prevent the locking plate **124** from rotating therein. That is, the locking plate **124** has a shape that prevents it from rotating within the cavity **142** when the locking plate **124** is inserted into the cavity **142**.

It will be further appreciated, moreover, that the mating slot **154** has dimensions that relate to the shape of the threaded shaft **108**. As indicated above, a majority of the threaded shaft **108** defines an externally threaded cylinder **130** with a pair of oppositely facing flat surfaces **132**. The mating slot **154**, in turn, is characterized by a pair of opposed sidewalls **176**, which are spaced apart by a distance about equal to a spacing between the pair of oppositely facing flat surfaces **132**. At the same time the mating slot **154** is further characterized by a pair of opposite sidewalls **178** spaced apart by a distance greater than the maximum diameter of the externally threaded cylinder **130**. Configured in this manner, the threaded shaft **108** is able to be translated longitudinally through the mating slot **154**, but the threaded shaft **108** is not able to rotate inside the mating slot **154** because of interference between the pair of oppositely facing flat surfaces **132** and the pair of opposed sidewalls **176**. Stated another way, the threaded shaft **108** is so

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shaped as to cooperate with the mating slot **154** to prevent the threaded shaft **108** from rotating independently of the locking plate **124** while still being translatable in the longitudinal direction therethrough.

Lastly, it will be noted that the cavity **142** defines several internal shoulders **180** therein. In the present illustrative embodiment, these internal shoulders **180** are located at the four corners of the rectangular cavity **142**.

FIGS. 5-7 show partially cutaway elevational views of the hook assembly **104** with the purpose of showing how the elements cooperate to allow the hook orientation to be changed and locked. As will be shown by this sequence of figures, the hook assembly **104** is operative to take on a plurality of locked states and an unlocked state. Each of the plurality of locked states is characterized by the locking plate **124** being pressed into the cavity **142** by the biasing element **126** and thereby prevented from rotating by the locking body **120** in the manner set forth above. With the locking plate **124** unable to rotate, the threaded shaft **108** and the hook **106** are likewise unable to rotate about the longitudinal axis. In contrast, the unlocked state is characterized by the locking plate **124** being outside the cavity **142** and thereby free to rotate, along with the threaded shaft **108** and the hook **106**.

The present hook assembly **104** is operative to take on four different locked states, each of which is characterized by a different orientation of the hook **106** relative to the frame **102**. FIG. 5 shows the hook assembly **104** in a representative one of these four locked states. In this locked state, the biasing element **126** presses the locking plate **124** against the threaded receiver **122** and the internal shoulders **180**, and, in response, the protruding lip **138** of the threaded receiver **122** abuts a perimeter **182** of the aperture **144** inside the cavity **142**. The locking plate **124** thereby occupies the cavity **142** of the locking body **120**, causing the locking plate **124** to be restricted in rotation. In contrast, the threaded receiver **122** remains free to rotate inside the cavity **142** in response to the rotation of the tensioning wheel **110**. Rotating the tensioning wheel **110** and the threaded receiver **122** causes the threaded shaft **108** and the hook **106** to be translated along the longitudinal axis of the apparatus **100**. The apparatus **100** thereby retains its "tensioning" functionality while in the locked state. Stated another way, the tensioning wheel **110** may be rotated with the apparatus **100** in the locked state to cause the hook **106** to move in the longitudinal direction relative to the frame **102** while maintaining a fixed orientation relative to the frame **102**.

FIG. 6 shows the apparatus **100** in the unlocked state. The unlocked state can be achieved from one of the locked states by manually pressing the threaded receiver **122** against the locking plate **124** so as to translate the locking plate **124** out of the cavity **142** counter to the bias of the biasing element **126**. The manual pressing force may be applied to the tensioning wheel **110** because, as discussed above, the tensioning wheel **110** is translatable coupled to the threaded receiver **122**. In the unlocked state, the biasing element **126** is compressed and the protruding lip **138** is in spaced relation to the perimeter **182** of the aperture **144**, but still partially occupies the cavity **142**. With the locking plate **124** outside the locking body **120**, the locking plate **124** and, correspondingly, the threaded shaft **108** and the hook **106** may be rotated relative to the frame **102**. In FIG. 6, it will be observed that, for example, the hook **106** is rotated by about 45-degrees relative to its orientation in FIG. 5.

Once the hook **106**, the threaded shaft **108**, and the locking plate **124** have achieved an orientation where the locking plate **124** can be reinserted into the locking body **120** (in the present embodiment, at 0-degrees, 90-degrees, 180-degrees,

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and 270-degrees), the manual force can be relieved on the tensioning wheel **110** so as to allow the apparatus **100** to return to a locked state in response to the biasing force from the biasing element **126**. Such a condition is shown in FIG. 7, which shows the locking plate **124** in an orientation where reinsertion into the cavity **142** of the locking body **120** is allowed, but before the manual downward force has actually been relieved. The hook **106** can thereby be quickly and easily swiveled between, and locked into, multiple orientations relative to the frame **102** without the use of tools. The swiveling may occur in either direction (i.e., clockwise or counterclockwise). In any one of the locked states, moreover, the tensioning function that incorporates the longitudinal translation of the hook **106** relative to the frame **102** remains available to the user through the rotation of the tensioning wheel **110**.

An additional feature of the apparatus **100** is the tensioning wheel locking mechanism **156**, briefly mentioned above. The main components of the tensioning wheel locking mechanism **156** are best seen in FIG. 4, while the positioning and functioning of the tensioning wheel locking mechanism **156** in the apparatus **100** are best seen in the partially cutaway front perspective view in FIG. 8. The tensioning wheel locking mechanism **156** includes a button **184**, a coil spring **186**, and a retaining pin **188**. The button **184** defines an extension **190** at one end. The retaining pin **188** fixes the tensioning wheel locking mechanism **156** to an outside sidewall of the locking body **120** via two tabs **192** with aligned holes. So placed, the coil spring **186** pushes the extension-end of the button **184** outward away from the locking body **120** towards a cylindrical sidewall **194** of the tensioning wheel **110**. The cylindrical sidewall **194** defines a plurality of locking windows **196** arranged along its perimeter.

When the tensioning wheel **110** is turned in a direction that increases tension in the apparatus **100** (in this embodiment, in the counter-clockwise direction), the extension **190** on the button **184** continuously ratchets into and out of the locking windows **196** as they move past. A user may therefore increase tension without attention to the tensioning wheel locking mechanism **156**. On the other hand, once a given tension is achieved, the button **184** interferes with the tensioning wheel **110** being turned in the opposite direction. That is, the extension **190** sits within one of the locking windows **196** and will abut against an inside edge of that locking window **196** if the user tries to rotate the tensioning wheel **110** so as to relieve tension (the condition shown in FIG. 8). This locked state may only be defeated by having the user manually press the button **184** in against the force of the coil spring **186** to move the button **184** and its extension **190** out of the way.

The tensioning wheel locking mechanism **156** thereby allows tension to be increased via the tensioning wheel **110** without interference, but requires manual intervention in the form of a button press to reduce tension via the tensioning wheel **110**. This assures that the tension on a secured load is not inadvertently reduced, which may allow the load to shift.

Once understood from the details provided herein, the elements of the hook assembly **104** may be formed utilizing conventional manufacturing techniques or, in the case of fasteners, springs, and the like, sourced commercially. For strength, most of the elements of the apparatus **100** preferably comprise a metal such as, but not limited to, steel. Metal machining is described in W. A. Knight et al., *Fundamentals of Metal Machining and Machine Tools*, Third Edition, CRC Press, 2005, which is hereby incorporated by reference herein.

The hook assembly **104** lends itself to several modifications. These numerous alternative embodiments will be apparent to one skilled in the art given the teachings herein.

For example, as detailed above, the illustrative apparatus **100** utilizes a geometric relationship between the cavity **142** of the locking body **120** and the locking plate **124** that provides four different locked orientations of the hook **106** relative to the frame **102** with 90-degree intervals therebetween. Nevertheless, modified geometries allow additional locked orientations to be designed into an apparatus, as desired. Use of a hexagonal locking plate in combination with a rectangular cavity (like the cavity **142**), for example, may allow an alternative embodiment of the invention to take on six locked hook orientations with 60-degree intervals. Similarly, an octagonal locking plate may allow another alternative embodiment of the invention to take on eight locked geometries with 45-degree intervals. Thus progressively more locked orientations may become available as more outside edges are added to the locking plate. At the same time, other modified geometries may allow fewer than four locked orientations. For example, a rectangular locking plate in combination with a rectangular cavity may provide only two locked hook orientations with 180-degree intervals. Use of a triangular locking plate in combination with a triangular cavity (instead of rectangular) may provide three locked orientations with 120-degree intervals. Accordingly, there are a multiplicity of related geometries that may be used for a locking plate and a locking body, and these alternative embodiments would also come within the scope of the invention.

Moreover, while the apparatus **100** is described as having a hook **106** at one end, alternative forms of attachment elements may be utilized. Alternative embodiments may utilize, for example, attachment elements such as, but not limited to, rings or devises.

Even further, while the above-described apparatus **100** utilizes particular fasteners such as the roll pins **158** and the bolts **164**, it should be reinforced that these particular fasteners are merely by way of illustration, and that other means of fixation may be used in their place. In one or more alternative embodiments falling within the scope of the invention, for example, screws, an adhesive, or joining may be utilized to fixate one or both of the locking body **120** and the biasing element **126** to the frame **102**.

Finally, while the biasing element **126** in the illustrative apparatus **100** is in the form of a leaf spring, alternative biasing elements may be utilized. In one or more alternative embodiments, the leaf spring may, for example, be replaced by two double-coil springs that are fixed to the frame **102** by bolts **164** and function to press upward on the locking plate **124**. Once the substitution is made, overall functionality will remain similar to what was detailed before.

## II. Chain Capture Assembly

FIG. 9 shows a rear perspective view of the chain capture assembly **112** in the apparatus **100**, in accordance with an illustrative embodiment of the invention. FIG. 10, moreover, shows a front exploded perspective view of the elements in FIG. 9. The chain capture assembly **112** comprises: a primary elongate member **900**, a secondary elongate member **902**, the release handle **904**, a release handle label plate **906**, a first locking spring **908**, a second locking spring **910**, a first spacer **912**, a second spacer **914**, a first rotatable member **916**, a second rotatable member **918**, a chain keeper member **920**, a coil spring **922**, a third elongate member **924**, and a chain capture block **926**.

In the present illustrative embodiment, the primary elongate member **900** describes a cylinder, which is fixed to the frame **102** by a bolt **928** that is terminated by a nut **930** (FIG.

1). In spanning between the pair of parallel sidewalls **103** of the frame **102**, the primary elongate member **900** passes through: the release handle **904**, the first spacer **912**, the first rotatable member **916**, the chain keeper member **920**, the coil spring **922**, the second rotatable member **918**, and the second spacer **914**. In so doing, the primary elongate member **900** pivotally supports the release handle **904**, the first and second rotatable members **916**, **918**, and the chain keeper member **920**. At the same time, the secondary elongate member **902** also spans between the pair of parallel sidewalls **103** of the frame **102**. In the present illustrative embodiment, the secondary elongate member **902** describes a pin that is held in place by cotter pins **932** (FIGS. 1 and 2). The secondary elongate member **902** rides in a first shaped guide slot **934** and a second shaped guide slot **936** in the release handle **904**. The first and second shaped guide slots **934**, **936** are each characterized by a respective arc **938** with an angled projection **940** at one end.

The first and second rotatable members **916**, **918** are disposed in spaced relation to one another. The first rotatable member **916** defines a first slot **942** that engages a lower edge **946** of the release handle **904**, while the second rotatable member **918** defines a second slot **944** that engages the same lower edge **946**. Configured in this manner, the first and second rotatable members **916**, **918** are pivotally coupled to the release handle **904** so that the first and second rotatable members **916**, **918** pivot in unison with the release handle **904**.

The third elongate member **924** is supported between the first and second rotatable members **916**, **918** via respective holes **948**, **950** in the first and second rotatable members **916**, **918**. Positioned in this manner, the third elongate member **924** describes an arc in response to the pivoting of the first and second rotatable members **916**, **918**. The coil spring **922** spans between the chain keeper member **920** and the third elongate member **924**. This coupling urges the chain keeper member **920** to pivot in unison with the first and second rotatable members **916**, **918**.

The chain capture block **926** is pivotally mounted to the frame **102** via two cylindrical bosses **952**, **954** that engage matching holes in the pair of parallel sidewalls **103** of the frame **102**. In the present non-limiting embodiment, the chain capture block **926** defines a cavity **956** shaped so as to accept two interlocked links of the chain **114**. The two links are in substantially normal relation with one another.

Finally, the first and second locking springs **908**, **910** span between capture holes **958** in the release handle **904** and the secondary elongate member **902**. The first and second locking springs **908**, **910** place a downward bias on the release handle **904**. This downward bias urges the release handle **904** in the direction of the chain capture block **926**. The release handle label plate **906** is riveted to the release handle **904** via rivets **960**. The release handle label plate **906** may contain written information such as manufacturer name, part number, date of manufacture, abbreviated instructions for use, any limitations or warnings, and the like.

In use, the first and second shaped guide slots **934**, **936** in the release handle **904** cooperate with the secondary elongate member **902** to restrict the motion of the release handle **904** between three configurations: a locked configuration, an unlocked configuration, and an open configuration. These configurations are shown in FIGS. 11-13. More particularly, FIGS. 11-13 show a sequence of partially broken, side elevational views of the chain capture assembly **112** and the chain **114** while the chain release handle **904** is in the locked configuration, the unlocked configuration, and the open configuration, respectively.

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In the locked configuration (FIG. 11), the secondary elongate member 902 occupies the respective angled projections 940 of the first and second shaped guide slots 934, 936, and the release handle 904 sits flush (i.e., vertically) in the frame 102. In this configuration, the release handle 904 may not be pivoted relative to the frame 102. At the same time, a first abutment surface 962 of the first rotatable member 916 abuts a first landing surface 964 of the chain capture block 926, while a second abutment surface 966 of the second rotatable member 918 abuts a second landing surface 968 of the chain capture block 926. The first and second rotatable members 916, 918 thereby hold the chain capture block 926 in a fixed orientation in the frame 102 with the mouth of the cavity 956 angled somewhat upward. This orientation of the chain capture block 926 is hereinafter called the “chain capture orientation.”

A user may insert the chain 114 into the chain capture block 926 while the release handle 904 is in the locked configuration and the chain capture block 926 is in the chain capture orientation simply by inserting two links of the chain 114 into the cavity 956 of the chain capture block 926 while the chain 114 is not under tension. In making this insertion, the vertically oriented link of the two chain links will press against a projection 970 on the chain keeper member 920, but the chain keeper member 920 will pivot out of the way so that the vertical link can pass unimpeded. After the vertical chain link passes, the chain keeper member 920 will snap back into position in response to the bias of the coil spring 922.

In contrast, once the chain links of the chain 114 are inserted into the cavity 956 of the chain capture block 926 (FIG. 11), the projection 970 of the chain keeper member 920 will interfere with the removal of those links from the chain capture block 926. More particularly, tabs 972 on the chain keeper member 920 will interact with the third elongate member 924 to stop the chain keeper member 920 from pivoting in the required direction for the chain 114 to be removed. The chain keeper member 920 therefore allows easy insertion of the chain 114 into the chain capture block 926, but interferes with the removal of the chain 114 once so inserted. The chances of the chain 114 inadvertently falling out of the chain capture block 926 are thereby mitigated.

Once the chain 114 is inserted into the chain capture block 926 and under tension, the chain capture block 926 in cooperation with the remainder of the chain capture assembly 112 will securely capture the chain 114. This configuration will be maintained until a user chooses to intentionally release the chain 114. To release the chain 114 requires that two distinct forces be applied to the release handle 904.

The first step in releasing the chain 114 is shown in FIG. 12, which shows the release handle 904 after it has been translated from the locked configuration into the unlocked configuration. To achieve this unlocked configuration, the release handle 904 is translated relative to the frame 102 upward away from the chain capture block 926 while the orientation of the release handle 904 relative to the frame 102 remains largely fixed (i.e., the release handle 904 remains flush to the frame 102). This upward motion occurs against the bias of the first and second locking springs 908, 910, thereby requiring an intentional actuation by the user. That is, the first and second locking springs 908, 910 urge the release handle 904 from the unlocked configuration back into the locked configuration when the release handle 904 is translated upward into the unlocked configuration. Once in the unlocked state, the secondary elongate member 902 has moved from the respective angled projections 940 of the first and second shaped guide slots 934, 936 into their respective arcs 938. The release handle 904 therefore is no longer restricted from

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pivoting, as was the case in the locked configuration (FIG. 11). Nevertheless, as long as the release handle 904 stays flush to the frame 102, the chain capture block 926 remains restricted in its ability to pivot in the frame 102, and the chain 114 remains secured.

Subsequently, in a second separate, positive motion, the release handle 904 may be pivoted relative to the frame 102 to achieve the open configuration in FIG. 13. In the open configuration, the first and second abutment surfaces 962, 966 of the first and second rotatable members 916, 918 disengage from the first and second landing surfaces 964, 968 of the chain capture block 926, allowing the chain capture block 926 to now freely pivot within the frame 102. As shown in FIG. 13, the chain capture block 926 pivots in the frame 102 to a “chain release orientation.” The chain 114 is thereby allowed to fall away from the chain capture assembly 112 and, more generally, the apparatus 100.

At the same time, when the first and second rotatable members 916, 918 pivot from the locked and unlocked configuration (FIGS. 11 and 12) to the open configuration (FIG. 13), the third elongate member 924 describes a short arc. This arcuate motion is shown in the partially broken side elevational view in FIG. 14. Motion of the third elongate member 924 causes the chain keeper member 920 to also pivot in association with the first and second rotatable member 916, 918, visible by the motion of the tab 970 in FIGS. 11-14.

Subsequently, to place the apparatus 100 in condition to again receive the chain 114, the chain capture block 926 is returned to the chain capture orientation and the release handle 904 is pivoted back into its unlocked configuration flush to the frame 102 (FIG. 12). Thereafter, the release handle 904 is allowed to translate downward into the locked configuration (FIG. 11).

To summarize, the chain capture block 926 is operative to achieve a chain capture orientation and a chain release orientation relative to the frame 102. The first and second shaped guide slots 934, 936 in the release handle 904 cooperate with the secondary elongate member 902 to allow the release handle 904 to be translated relative to the frame 102 between the locked configuration and the unlocked configuration. At the same time, the first and second shaped guide slots 934, 936 further cooperate with the secondary elongate member 902 to allow the release handle 904 to be pivoted relative to frame 102 between the unlocked configuration and the open configuration. With the release handle 904 in the locked and unlocked configurations, the first and second rotatable members 916, 918 interfere with the pivoting of the chain capture block 926 from the chain capture orientation to the chain release orientation. With the release handle 904 in the open configuration, the chain capture block 926 is free to pivot between the chain capture orientation and the chain release orientation.

Configured in the manner set forth above, the chain capture assembly 112 requires that two distinct forces be applied to the release handle 904 in order to release the chain 114. Prior art assemblies, in contrast, have typically required only a single motion of a release handle to release a chain, making these prior art assemblies more prone to inadvertent releases. Advantageously, the two separate forces required to release the chain capture assembly 112 (i.e., first the upward translation of the release handle 904, and then the pivoting of the release handle 904 away from the frame 102) are in two almost opposite directions. Inadvertent releases are therefore extremely unlikely when utilizing the chain capture assembly 112, and more generally, embodiments in accordance with aspects of the invention.

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Once understood from the details provided herein, the elements of the chain capture assembly **112** may be formed utilizing conventional manufacturing techniques or, in the case of fasteners, springs, and the like, sourced commercially. For strength, most of the elements of the apparatus **100** preferably comprise a metal such as, but not limited, to steel. Metal machining is described in W. A. Knight et al., *Fundamentals of Metal Machining and Machine Tools, Third Edition* (cited earlier).

Like the hook assembly **104**, the chain capture assembly **112** lends itself to several modifications. As just one example, the particular fasteners and types of springs utilized in the embodiment set forth above may be replaced by equally suitable fasteners and biasing elements, and the results would still come within the scope of the invention. As another example, embodiments in accordance with aspects of the invention may utilize only a single rotatable member (instead of a pair of rotatable members) to obtain the desired functionality. These numerous alternative embodiments will be apparent to one skilled in the art given the teachings herein.

All the features disclosed herein may be replaced by alternative features serving the same, equivalent, or similar purposes, unless expressly stated otherwise. Thus, unless expressly stated otherwise, each feature disclosed is one example only of a generic series of equivalent or similar features.

Any element in a claim that does not explicitly state “means for” performing a specified function or “step for” performing a specified function is not to be interpreted as a “means for” or “step for” clause as specified in AIA 35 U.S.C. §112(f). In particular, the use of “step of” in the claims herein is not intended to invoke the provisions of AIA 35 U.S.C. §112(f).

What is claimed is:

**1.** An apparatus comprising:

- a frame;
- a primary elongate member, the primary elongate member fixated to the frame;
- a release handle, the release handle pivotally supported by the primary elongate member, and defining a shaped guide slot;
- a secondary elongate member, the secondary elongate member fixated to the frame, and riding in the shaped guide slot;
- a rotatable member, the rotatable member pivotally supported by the primary elongate member, and pivotally coupled to the release handle such that the rotatable member pivots in unison therewith; and
- a chain capture block, the chain capture block defining a cavity therein shaped to accommodate one or more links of a chain, and pivotally mounted to the frame such that the chain capture block can achieve a chain capture orientation and a chain release orientation relative to the frame;

wherein:

- the shaped guide slot and the secondary elongate member cooperate to allow the release handle to be translated relative to the frame between a locked configuration and an unlocked configuration;
- the shaped guide slot and the secondary elongate member cooperate to allow the release handle to be pivoted relative to the frame between the unlocked configuration and an open configuration;
- the rotatable member interferes with the pivoting of the chain capture block from the chain capture orientation to the chain release orientation when the release handle is in the locked or unlocked configuration; and

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the chain capture block is free to pivot between the chain capture orientation and the chain release orientation when the release handle is in the open configuration.

**2.** The apparatus of claim **1**, wherein an orientation of the release handle with respect to the frame remains fixed when the release handle is translated relative to the frame between the locked configuration and the unlocked configuration.

**3.** The apparatus of claim **1**, wherein the release handle is translated relative to the frame away from the chain capture block when the release handle is translated from the locked configuration to the unlocked configuration.

**4.** The apparatus of claim **1**, wherein the frame defines a pair of parallel sidewalls.

**5.** The apparatus of claim **4**, wherein the primary elongate member spans between the pair of parallel sidewalls.

**6.** The apparatus of claim **4**, wherein the secondary elongate member spans between the pair of parallel sidewalls.

**7.** The apparatus of claim **1**, wherein the shaped guide slot defines an elongate opening with a shorter opening projecting at an angle from one end.

**8.** The apparatus of claim **7**, wherein the secondary elongate member occupies the elongate opening when the release handle is in the unlocked configuration or the open configuration.

**9.** The apparatus of claim **7**, wherein the secondary elongate member occupies the shorter opening when the release handle is in the locked configuration.

**10.** The apparatus of claim **1**, wherein:  
the rotatable member defines a slot; and  
the slot engages the release handle.

**11.** The apparatus of claim **1**, wherein:  
the rotatable member defines an abutment surface;  
the chain capture block defines a landing surface; and  
the abutment surface contacts the landing surface when the chain capture block is in the chain capture orientation and the release handle is in the locked or unlocked configuration.

**12.** The apparatus of claim **1**, wherein:  
the frame defines two apertures; and  
the chain capture block comprises two bosses, the two bosses inserted into the two apertures.

**13.** The apparatus of claim **1**, further comprising a locking spring, the locking spring spanning between the secondary elongate member and the release handle.

**14.** The apparatus of claim **13**, wherein the locking spring urges the release handle from the unlocked configuration into the locked configuration when the release handle is in the unlocked configuration.

**15.** The apparatus of claim **1**, further comprising a second rotatable member, the second rotatable member pivotally supported in spaced relation to the rotatable member by the primary elongate member, and pivotally coupled to the release handle such that the second rotatable member pivots in unison therewith.

**16.** The apparatus of claim **15**, wherein the second rotatable member interferes with the pivoting of the chain capture block from the chain capture orientation to the chain release orientation when the release handle is in the locked or unlocked configuration.

**17.** The apparatus of claim **15**, further comprising:  
a chain keeper member, the chain keeper member pivotally supported by the primary elongate member and defining a projection;  
a third elongate member, the third elongate member spanning between the rotatable member and the second rotatable member such that the third elongate member

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describes an arc in response to the pivoting of the rotatable member and the second rotatable member; and a spring, the spring spanning between the chain keeper member and the third elongate member.

**18.** The apparatus of claim **17**, wherein the spring urges the chain keeper member to pivot in unison with the rotatable member and the second rotatable member via the third elongate member. 5

**19.** The apparatus of claim **17**, wherein the projection interferes with removal of the chain from the chain capture block when the chain capture block is in the chain capture orientation. 10

**20.** The apparatus of claim **1**, wherein the apparatus further comprises a hook.

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